Reg. No. :

# **Question Paper Code: 31571**

B.E. / B.Tech. DEGREE EXAMINATION, NOVEMBER 2015

Fifth Semester

Mechanical Engineering

# 01UME501 - DYNAMICS OF MACHINERY

(Regulation 2013)

Duration: Three hours

Maximum: 100 Marks

Answer ALL Questions

PART A - (10 x 2 = 20 Marks)

- 1. Define applied and constraint force.
- 2. Why flywheels are needed in forging and pressing operation?
- 3. Can a single cylinder engine be fully balanced? Why?
- 4. State the conditions for static and dynamic balancing.
- 5. Define critical or whirling speed.
- 6. What are the different types of vibrations?
- 7. What is the vibration isolation?
- 8. Define transmissibility.
- 9. How governors are classified?
- 10. What is gyroscopic torque?

## PART - B (5 x 16 = 80 Marks)

- 11. (a) A horizontal steam engine running at 120 *r.p.m.* has a bore of 250 *mm* and a stroke of 400 *mm*. The connecting rod is 0.6 *m* and mass of the reciprocating parts is 60 *kg*. When the crank has turned through an angle of  $45^{\circ}$  from the inner dead centre, the steam pressure on the cover end side is  $550 \text{ kN/m}^2$  and that on the crank end side is  $70 \text{ kN/m}^2$ . Considering the diameter of the piston rod equal to 50 mm, determine:
  - (i) turning moment on the crank shaft,
  - (ii) thrust on the bearings,
  - (iii) acceleration of the flywheel, if the power of the engine is 20 kW, mass of the flywheel 60 kg and radius of gyration 0.6 m. (16)

## Or

- (b) A single cylinder double acting steam engine develops 150 kW at a mean speed of 80 *r.p.m.* The coefficient of fluctuation of energy is 0.1 and the fluctuation of speed is  $\pm 2\%$  of mean speed. If the mean diameter of the flywheel rim is 2 *meter* and the hub and spokes provide 5% of the rotational inertia of the flywheel, find the mass and cross-sectional area of the flywheel rim. Assume the density of the flywheel material (which is cast iron) as 7200 kg/m<sup>3</sup>. (16)
- 12. (a) A shaft carries four masses A, B, C and D of magnitude 200 kg, 300 kg, 400 kg and 200 kg respectively and revolving at radii 80 mm, 70 mm, 60 mm and 80 mm in planes measured from A at 300 mm, 400 mm and 700 mm. The angles between the cranks measured anticlockwise are A to B 45°, B to C 70° and C to D 120°. The balancing masses are to be placed in planes X and Y. The distance between the planes A and X is 100 mm, between X and Y is 400 mm and between Y and D is 200 mm. If the balancing masses revolve at a radius of 100 mm, find their magnitudes and angular positions. (16)

#### Or

- (b) The following data refer to two cylinder locomotive with cranks at 90°, reciprocating mass per cylinder = 300 kg, crank radius = 0.3 m, driving wheel diameter =1.8 m, distance between cylinder centre lines = 0.65 m, distance between the driving wheel central planes =1.55m. Determine:
  - (i) the fraction of the reciprocating masses to be balanced, if the hammer blow is not to exceed 46 *kN* at 96.5 *km.p.h*.
  - (ii) the variation in tractive effort,
  - (iii) the maximum swaying couple.

(16)

- 13. (a) The measurements on a mechanical vibrating system show that it has a mass of 8 kg and that the springs can be combined to give an equivalent spring of stiffness 5.4 *N/mm*. If the vibrating system have a dashpot attached which exerts a force of 40 *N* when the mass has a velocity of 1m/s, find:
  - (i) Critical damping coefficient
  - (ii) Damping factor
  - (iii) Logarithmic decrement
  - (iv) Ratio of two consecutive amplitudes. (16)

#### Or

- (b) A steel shaft  $1.5m \log is 95mm$  in diameter for the first 0.6m of its length, 60mm in diameter for the next 0.5m of the length and 50mm in diameter for the remaining 0.4m of its length. The shaft carries two flywheels at two ends, the first having a mass of 900 kg and 0.85m radius of gyration located at the 95mm diameter end and the second having a mass of 700 kg and 0.55m radius of gyration located at the other end. Determine the location of the node and the natural frequency of free torsional vibration of the system. The modulus of rigidity of shaft material may be taken as  $80 GN/m^2$ . (16)
- 14. (a) A mass of 10 kg is suspended from one end of a helical spring, the other end being fixed. The stiffness of the spring is 10 *N/mm*. The viscous damping causes the amplitude to decrease to one-tenth of the initial value in four complete oscillations. If a periodic force of 150 cos 50 t N is applied at the mass in the vertical direction, find the amplitude of the forced vibrations. What is its value of resonance? (16)

#### Or

- (b) A machine part of mass 2 kg vibrates in a viscous medium. Determine them damping coefficient when a harmonic exciting force of 25 N results in a resonant amplitude of 12.5 mm with a period of 0.2 s. If the system is excited by a harmonic force of frequency 4 Hz what will be the percentage increase in the amplitude of vibration when damper is removed as compared with that with damping. (16)
- 15. (a) In an engine governor of the porter type, the upper and lower arms are 200mm and 250 mm respectively and pivoted on the axis of rotation. The mass of the central load is 15 kg, the mass of each ball is 2 kg and friction of the sleeve together with the resistance of the operating gear is equal to a load of 25 N at the sleeve. If the limiting inclinations of the upper arms to the vertical are  $30^{\circ}$  and  $40^{\circ}$ , find taking friction into account and range of speed of the governor. (16)

- (b) The turbine rotor of a ship has a mass of  $3500 \ kg$ . It has a radius of gyration of  $0.45 \ m$  and a speed of  $3000 \ r.p.m$ . clockwise when looking from stern. Determine the gyroscopic couple and its effect upon the ship:
  - (i) When the ship is steering to the left on a curve of 100 m radius at a speed of 36 km/h.
  - (ii) When the ship is pitching in a simple harmonic motion, the bow falling with its maximum velocity. The period of pitching is 40 s and the total angular displacement between the two extreme positions of pitching is 12 *degrees*. (16)

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