A		Reg. No. :						
Question Paper Code: 55703								
B.E. / B.Tech. DEGREE EXAMINATION, NOV 2018								
Fifth Semester								
Mechanical Engineering								
15UME503 - DESIGN OF MACHINE ELEMENTS								
(Regulation 2015)								
(Approved data book are permitted)								
Duration: Three hoursMaximum: 100 MarksAnswer ALL Questions								
PART A - $(10 \text{ x } 1 = 10 \text{ Marks})$								
1.	Ability of a material to undergo large permanent deformation in CO1- R tension is called							
	(a) Toughness	(b) Stiffness	(c) Plasticity	(d) Hardness				
2.	Select the theory of failure for an aluminum component under steady CO1- loading							
	(a) Principal stress theory		(b) Principal strain theory					
	(c) Strain energy theory		(d) Maximum shear stress theory					
3.	The angle of twist for a transmission shaft is inversely proportional to CO2- I							
	(a) Shaft diameter	(b) Shaft diameter ²	(c) Shaft diameter ³	(d) Shaft diameter ⁴				
4.	A key connecting a flange coupling to a shaft is likely to fail in CO2- R							
	(a) Shear	(b) Tension	(c) Torsion	(d) Bending				
5.	In fillet welded joint weakest area of weld is CO3- R							
	(a) Throat	(b) Root	(c) Face	(d) Toe				

6.	When a nut is tightened by placing a washer below it, the shank of CO bolt is subjected to							
	(a) Direct shear stress		(b) Torsional shear stress					
	(c) Tensile stress		(d) Compressive stress					
7.	When a helical spring is cut into two equal halves, stiffness of each of CO4- R the resulting spring will be							
	(a) Unaltered	(b) Double	(c) One half	(d) On	e fourth			
8.	The cross section of the flywheel arm(or spoke) is				CO4- R			
	(a) I section	(b) Rectangular	(c) Elliptical	(d) Cir	cular			
9.	In radial bearings, the	load acts			CO5- R			
	(a) Along the axis of	rotation	(b) Perpendicular to the axis of the rotation					
	(c) Parallel to the axis	s of rotation	(d) Inclined to the axis of rotation					
10.	Whipping stress in due to				CO5- R			
	(a) Vibration of crank shaft		(b) Reciprocating motion of piston					
	(c) Inertia force on co	onnecting rod	(d) Obliquity of connecting rod					
PART - B (5 x 2 = 10 Marks)								
11.	State the three modes of failure occur in mechanical components.							
12.	. List the reasons for using hollow shaft than solid shaft in transmission of power							
13.	. What is meant by caulking?							

- 14. State the reasons for using cast Iron flywheel. CO4- R
- 15. List the application areas of sliding bearing and rolling bearing. CO5- U

$$PART - C (5 \times 16 = 80 Marks)$$

16. (a) A transmission shaft carries a pulley midway between the two CO1- App (16) bearings. The bending moment at the pulley varies from 200Nm to 600Nm, as the torsional moment in the shaft varies from 70Nm to 200Nm. The frequencies of variation of bending and torsional moments are equal to the shaft speed. The shaft is made of steel $Fe400(S_{ut}=540N/mm^2 \text{ and } S_{yt}=400N/mm^2)$. The corrected endurance limit of the shaft is 200N/mm². Determine the diameter of the shaft using FOS of 2.

Or

- (b) A mass of 50Kg drops through 25mm at the center of a 250mm CO1- App (16) long simply supported beam. The beam has square cross section. It is made of steel 30C8 (S_{yt} =400N/mm²) and FOS is 2. The modulus of elasticity is 207GPa.Determine the dimension of the cross section of the beam.
- 17. (a) A shaft of 1m long is supported by two bearings 500mm apart. CO2- App (16) The pulley 1 of 250mm diameter is attached on the left extreme side of the shaft whereas the pulley 2 of 500mm diameter is attached on the right extreme side of the shaft. The masses of pulley 1 and 2 are 10 and 30kg respectively. The belt tensions act vertically downward and the ratio of belt tensions on the tight side to slack side of each pulley is 2.5:1. The shaft is made of plain carbon steel 40C8 (S_{yt} =380N/mm²) and the FOS is 3. Estimate suitable diameter of the shaft. If the permissible angle of twist is 0.5° per meter length, calculate the shaft diameter on the basis of torsional rigidity. G=79300N/mm².

Or

- (b) Design a muff coupling to connect two steel shafts transmitting CO2- App (16) 25kW power at 360rpm. The shafts and key are made of plain carbon steel 30C8 ($S_{yt}=S_{yc}=400N/mm^2$). The sleeve is made of grey cast iron FG200 ($S_{ut}=200N/mm^2$). The factor of safety for the shafts and key is 4. For sleeve, the factor of safety is 6 based on ultimate strength.
- 18. (a) It is required to design a knuckle joint to connect two circular CO3- App (16) rods subjected to an axial tensile force of 50kN. The rods are co-axial and small angular movement between their axes is permissible. Design the joint and specify the dimensions of its components and select suitable materials for the parts.

Or

(b) A steel plate, 100mm wide and 10mm thick is joined with another CO3- App (16) steel plate by means of a single transverse and double parallel fillet welds. The strength of the welded joint should be equal to the strength of the plates to be joined. The permissible tensile and shear stresses for the weld materials and the plates are 70 and 50N/mm2 respectively. Find the length of each parallel fillet weld. Assume tensile force acting on the plates as static.

19. (a) A railway wagon moving at a velocity of 1.5m/s is brought to rest CO4- App (16) by a bumper consisting of two helical springs arranged in parallel. The mass of the wagon is 1500kg. The springs are compressed by 150mm in bringing the wagon to rest. The spring index is 6. The springs are made of oil hardened and tempered steel wire (G=81370N/mm2). The constants A and m can be taken as 1855 and 0.187 respectively. The permissible shear stress is 50% of the ultimate stress. Design the spring.

Or

- (b) The turning moment diagram of a multi cylinder engine is drawn CO4- App (16) with a scale of (1mm=1°) on the abscissa and (1mm=250Nm) on the ordinate. The intercepted areas between the torque developed by the engine and the mean resisting torque of the machine, taken in order from one end are -350,+800,-600,+900,-550,+450 and -650mm². The engine is running at a mean speed of 750rpm and coefficient of speed fluctuations is limited to 0.02. A rimmed flywheel made of grey cast iron FG200 (density 7100kg/m³) is provided. The spokes, hub and shaft are assumed to contribute 10% of the required moment of inertia. The rim has rectangular cross section and the ratio of wide to thickness is 1.5. Determine the dimensions of the rim.
- 20. (a) Following data is given for a 360° hydrodynamic bearing: radial CO5- App (16) load is 3.2kN,journal speed is 1490rpm,l/d ratio is 1,unit bearing pressure is 1.3MPa,radial clearance is 0.05mm and viscosity of lubricant is 25cP. Assuming that the total heat generated in the bearing is carried by the total oil flow in the bearing. Design the bearing with flow requirement of 11iter/min.

Or

(b) A single row deep groove ball bearing is subjected to a radial CO5- App (16) force of 8kN and a thrust force of 3kN. The shaft rotates at 1200rpm. The expected life L_{10h} of the bearing is 20000hr. The minimum acceptable diameter of the shaft is 75mm. Select suitable ball bearing for this application.